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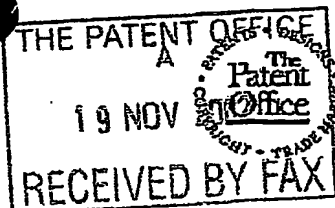
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19 NOV 2002

The Patent Office

Cardiff Road
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NP10 8QQ

1. Your reference

02445 GB

2. Patent application number
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0226940.5

3. Full name, address and postcode of the or each applicant (underline all surnames)

Hansen Transmissions International NV
Leonardo da Vincilaan 1
B-2650 Edegem
Antwerp

Patents ADP number (if you know it)

If the applicant is a corporate body, give the country/state of its incorporation

Belgium

635 444 3002

4. Title of the invention

Wind Turbine Gear Unit with Integrated Rotor Bearing

5. Name of your agent (if you have one)

"Address for service" in the United Kingdom
to which all correspondence should be sent
(including the postcode)Mr J R Badger
Invensys Intellectual Property
P O Box 8433
Redditch, B98 0DW
United Kingdom

Patents ADP number (if you know it)

849 455 1001

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Country

Priority application number
(if you know it)Date of filing
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7. If this application is divided or otherwise derived from an earlier UK application, give the number and the filing date of the earlier application

Number of earlier application

Date of filing
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8. Is a statement of inventorship and of right to grant of a patent required in support of this request? (Answer 'Yes' if:

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11. I/We request the grant of a patent on the basis of this application.

Signature

J. R. Badger

Date

19/11/02

12. Name and daytime telephone number of person to contact in the United Kingdom

J R Badger

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DUPLICATE

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WIND TURBINE GEAR UNIT WITH INTEGRATED ROTOR BEARING

This invention relates to a wind turbine gear unit with integrated rotor bearing and in particular, though not exclusively, to a wind turbine gear unit of the kind comprising a planetary gear stage.

Advantages of wind turbine gear units with integrated rotor bearings include low weight and fewer parts. Issues to solve are rotor to nacelle load paths that pass through different components of the gear unit (and may influence its functionality) and serviceability (and/or replaceability) of some of the gear unit parts including the main bearing and sealing system.

The invention described herein seeks to provide at least some of the advantages typical to integrated rotor bearing designs while offering solutions to some of the disadvantages linked to such designs.

In accordance with one of its aspects the present invention provides a gear unit for use in wind turbines of the kind having an integrated main rotor bearing and comprising a planetary gear stage, said integrated main rotor bearing being adapted to provide location for the rotor hub relative to the wind turbine nacelle and to locate the planet carrier relative to a low speed ring gear, wherein forces due to bending moments acting between the rotor hub and nacelle are transmitted substantially directly from the rotor hub to the nacelle via the main bearing and substantially none of said bending moments are transmitted to the nacelle via the ring gear of the planetary gear stage.

The invention further teaches that preferably also no part of the planet carrier which is substantially in the same axial plane as a plane containing the planet gears transmits any said bending movements to the nacelle.

Invention teaches in particular that preferably the outer ring(s) of the main rotor bearing is secured to the nacelle and that the rotor hub and planet carrier are secured to the inner ring(s) of the main bearing. Accordingly the main bearing serves two functions, firstly it serves as the rotor bearing and secondly it supports the planet carrier. In its role as rotor bearing it acts to transfer substantially all rotor forces and moments (but not driving torque) to the nacelle structure. Mounting the main bearing with the inner ring(s) thereof

supporting the planet carrier and fastening the outer ring(s) of the bearing directly to the nacelle (optionally via a main bearing cover or intermediate housing structure), results in the rotor forces and moments being transferred substantially directly to the nacelle via the main bearing.

The main bearing may be of a double taper roller bearing, for example arranged in an O configuration and having a split inner ring, but other types of main bearing such as those comprising cylindrical bearings may be employed.

The ring gear of the planetary gear stage may be secured to the nacelle via an intermediate housing structure. That intermediate housing structure may be secured directly to the nacelle or to the outer ring(s) of the main bearing.

Preferably but not necessarily the planet gears lie axially displaced relative to the main bearing as considered in the direction of the axis about which the planet carrier rotates. It is further preferred that the sun gear is of the type known as a floating sun gear which is able to move slightly in a direction perpendicular relative to its axis of rotation so as always to lie centrally of the planet gears and ensure that torque is transmitted substantially equally by each of the planet gears.

Preferably the ring gear of the planetary gear stage is selectively releasably secured relative to the nacelle. In consequence all parts of the gear unit except for the planet carrier and main bearing may be dismantled without the need to remove the rotor. (For this purpose it is taught that the planet carrier may be of a two part construction whereby one part of the planet carrier which supports the planet bearings and planet gears may be selectively removed from a second part of the planet carrier which is secured to the main bearing (e.g. the inner ring(s) of the main bearing).

The invention further teaches that the main bearing may be provided with a seal pack to inhibit leakage of lubricant from the main bearing, and that the seal pack may be selectively removable insitu from the main bearing. The seal pack may be releasably secured to an inner or outer ring of the main bearing and may be arranged to be displaceable axially such that replaceable seals of the seal pack may be inspected and if necessary replaced. Preferably use is made of a seal pack which is secured to the nacelle, i.e. is non rotatable, in which case the seals may be of a split type with the split in the seal(s) being

positioned at an uppermost position at which there is no substantial risk of lubricant leaking through the split.

Embodiments of the invention will now be described, by way of example only, with the reference to the accompanying diagrammatic drawings in which: -

Figure 1 is a sectional view of a wind turbine gear unit in accordance with a first embodiment of the invention;

Figure 2 is a view similar to that of Figure 1 in accordance with a second embodiment of the invention, and

Figure 3 shows in cross section part of the embodiment of Figure 1 with the bending moment force transmission path illustrated.

In accordance with a first embodiment of the invention Figure 1 shows a gear unit installed in a wind turbine nacelle [1] with the rotor hub [2] attached directly to the low speed planet carrier [3] of the gear unit. The main bearing [4], which locates the rotor hub relative to the nacelle as well as the planet carrier relative to the ring gear [13] of the gear unit, is mounted on the planet carrier.

The two inner rings [6] of the main bearing may be located by the retainer ring [7] which is in turn located between the rotor hub and planet carrier. In this embodiment the rotor bolts [8] pass through the retainer ring into threaded holes in the planet carrier.

The gear unit is attached to the nacelle with bolts [10] that pass through the main bearing cover [9] into threaded holes in the outer ring(s) [5] of the main bearing. An intermediate housing [11] is connected to the stationary outer ring(s) of the main bearing with bolts [12]. The threaded holes for these bolts may be either in the outer ring(s) of the main bearing or the main bearing cover. The intermediate housing serves to connect the rest of the gear unit to the stationary structure. This could be done by fastening it to the ring wheel [13] with bolts [14].

The low speed planet carrier [3] is of a two part construction comprising a first part [33] secured to the main bearing inner rings and second part [34] which carries the planet gear bogies and bearings, with the second part [34] being selectively releasable from the first part [33] by removing the retention bolts [35]. The planet carrier comprises three bogie units substantially

uniformly circumferentially spaced about the axis of rotation of the planet carrier, and three groups of retention bolts [35] are provided at three circumferentially spaced positions intermediate the planet bogies.

Oil used to lubricate the main bearing is kept from leaking out of the unit by the seal pack [15]. The seal pack may consist of a removable seal holder that could be formed to create a labyrinth and carry a radial lip seal(s), V-ring(s) or a combination of both. The seal lips may rub against the retainer ring [7], the main bearing inner ring(s) [6] or a combination of both.

Before the gear unit is installed in the wind turbine, the retainer ring [7] and main bearing cover [9] are held in place by bolts (not shown in Fig 1) that are installed as to not interfere with the mating faces of the nacelle [1] and rotor [3] function is taken over by bolts [8] and [10].

Figure 2 shows a second embodiment of the invention. In this embodiment the outer ring(s) [5] of the main bearing is mounted to the nacelle structure [1] via the intermediate housing [11]. This could be done by using long bolts [10A] that pass through the main bearing cover [9], the main bearing outer ring(s) [5] and intermediate housing [11] into threaded holes in the nacelle structure. Alternatively the gear unit could be bolted to the nacelle structure using bolts [10B] that are inserted from inside the nacelle, pass through the intermediate housing [11] and into threaded holes in the outer ring(s) of the main bearing.

In this case the intermediate housing [11] and the main bearing cover [9] are held in place by bolts (not shown in Fig 2) until the unit is installed in the wind turbine at which time their function is transferred to bolts [8] and [10A] or [10B].

In accordance with the present invention the main bearing serves as a bearing for rotation of the rotor hub relative to the nacelle and also provides support for the planet carrier of the planet gear stage. In the aforescribed embodiments this is obtained by securing the rotor hub and planet carrier relative to the inner ring of the main bearing whereas the ring gear of the planetary gear is secured relative to the nacelle, either directly or via the outer ring of the main bearing. Accordingly the main bearing is able to transfer all axial and bending moment forces substantially directly to the nacelle structure,

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and the planetary gear stage experiences substantially only the drive torque provided by the rotor hub. Figure 3 shows the Figure 1 type of construction in which the main bearing inner ring supports the planet carrier and the outer ring is secured directly to the nacelle (via the main bearing cover or intermediate housing), and in consequence the axial and bending moment forces exerted by the rotor hub are transferred substantially directly to the nacelle via the main bearing.

CLAIMS

1. A gear unit for use in wind turbines of the kind having an integrated main rotor bearing and comprising a planetary gear stage, said integrated main rotor bearing being adapted to provide location for the rotor hub relative to the wind turbine nacelle and to locate the planet carrier relative to a low speed ring gear, wherein forces due to bending moments acting between the rotor hub and nacelle are transmitted substantially directly from the rotor hub to the nacelle via the main bearing and substantially none of said bending moments are transmitted to the nacelle via the ring gear of the planetary gear stage.
2. A gear unit according to claim 1 wherein no part of the planet carrier which is substantially in the same axial plane as a plane containing the planet gears transmits any said bending moments to the nacelle.
3. A gear unit according to claim 1 or claim 2 wherein the or each outer ring of the main rotor bearing is secured to the nacelle, and the rotor hub and planet carrier are secured to the inner ring(s) of the main bearing.
4. A gear unit according to any one of the preceding claims wherein said main bearing is a double taper roller bearing.
5. A gear unit according to claim 4 wherein said double taper roller bearing comprises two series of taper rollers arranged in an O configuration.
6. A gear unit according to claim 4 or claim 5 wherein said double taper roller bearing comprises a split inner ring.
7. A gear unit according to any one of the preceding claims wherein the outer ring(s) of the main bearing is secured substantially directly to the nacelle via a main bearing cover.
8. A gear unit according to any one of claims 1 to 6 wherein the main bearing is secured substantially directly to the nacelle via an intermediate housing.
9. A gear unit according to claim 8 wherein said intermediate housing is secured directly to the nacelle or to the outer ring of the main bearing.
10. A gear unit according to any one of the preceding claims wherein the planet gears lie axially displaced relative to the main bearing as considered in the direction of the axis about which the planet carrier rotates.

11. A gear unit according to any one of the preceding claims wherein the ring gear of the planetary gear stage is selectively releasably secured to the nacelle whereby at least the planet gears and ring gear may be dismantled without the need to remove the rotor.
12. A gear unit according to claim 11 wherein all parts of the gear unit except at least one of the main bearing and the planet carrier or a part thereof may be dismantled without the need to remove the rotor.
13. A gear unit according to claim 11 or claim 12 wherein the planet carrier is of a two part construction comprising one part to support the planet gears, said one part being selectively releasably secured to a second part of the planet carrier which is secured substantially directly relative to the main bearing.
14. A gear unit according to any one of the preceding claims wherein the main bearing is provided with a seal pack arranged to be selectively removeable in situ from the main bearing.
15. A gear unit according to claim 14 wherein said seal pack is displaceable axially relative to the main bearing.
16. A gear unit according to claim 14 or claim 15 wherein said seal pack comprises a circumferentially discontinuous seal.
17. A gear unit according to claim 1 and substantially as herein before described with reference to the accompanying drawings.

FIGURE 1

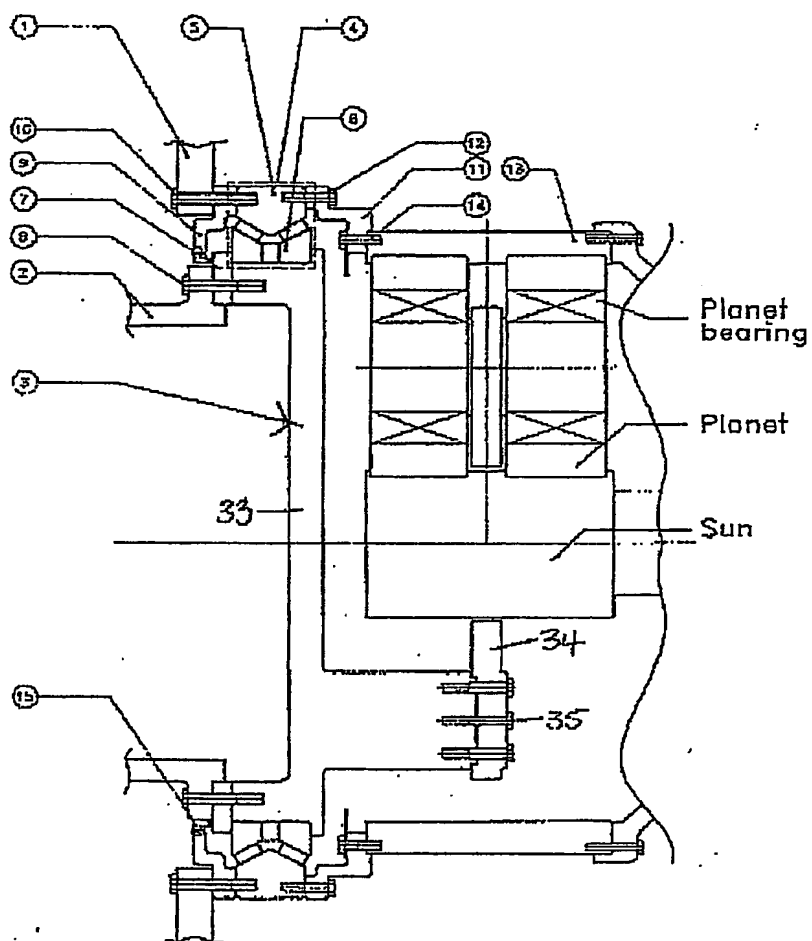


FIGURE 2

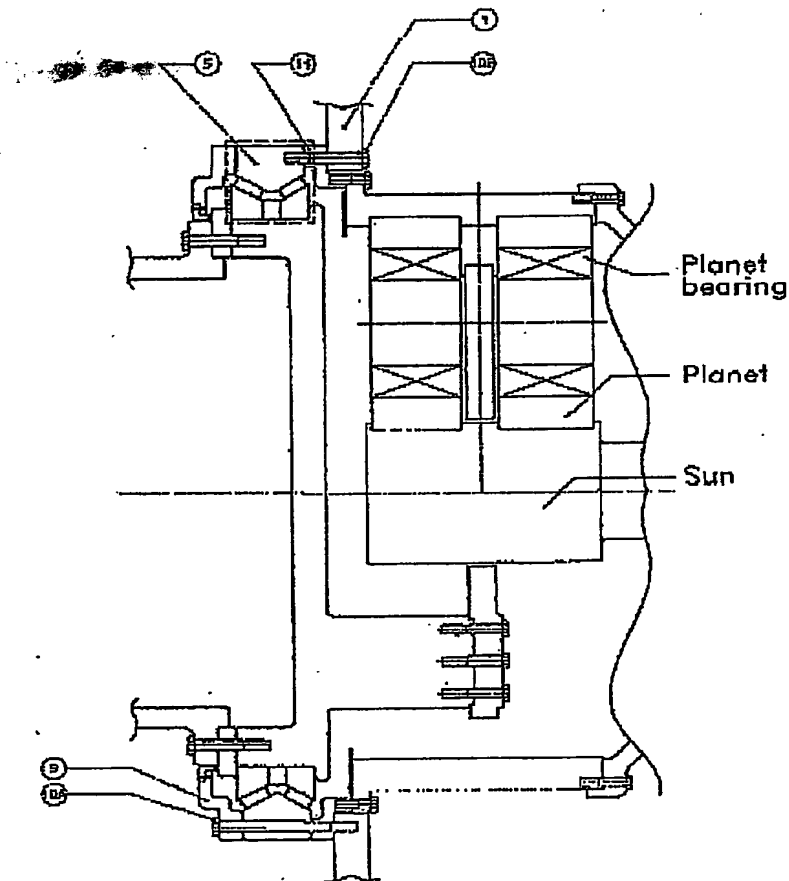
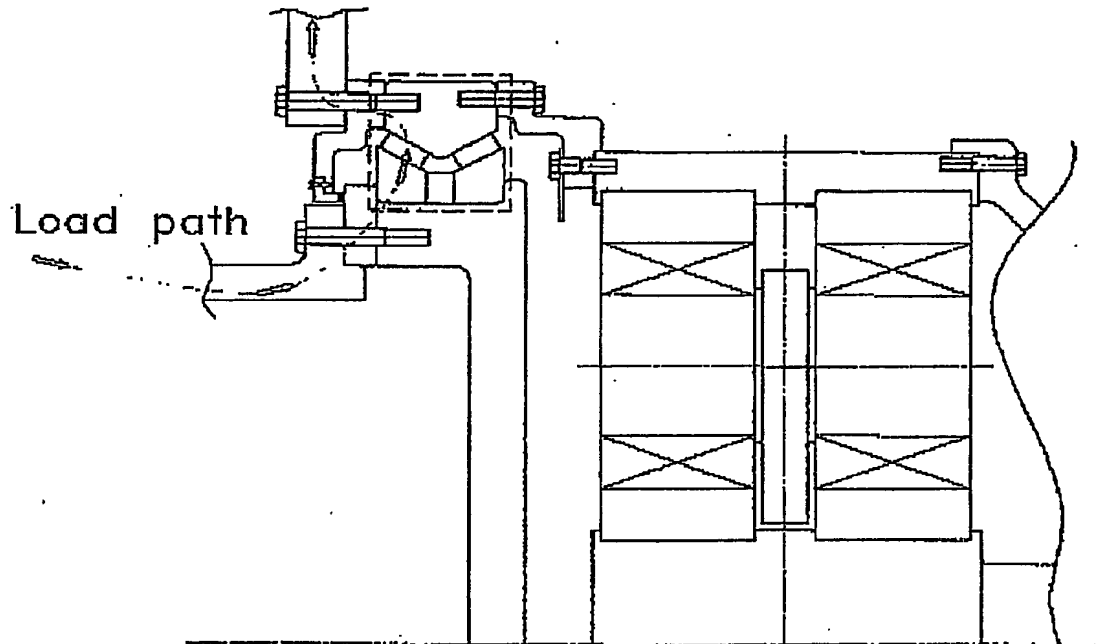


FIGURE 3



PCT Application
PCT/IB2003/006317



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